NUMERICAL SOLUTIONS OF FREE

CONVECTION FROM AN ISOTHERMAL INCLINED PLATE

A Thesis

Presented to

The Faculty of the Department of Mechanical Engineering University of Houston

In Partial Fulfillment

of the Requirments for the Degree Master of Science in Mechanical Engineering

by

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Chung-Jen Kau

August, 1967

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ABSTRACT

The differential equations decribing the free convection heat transfer about an inclined plate have been solved by a numerical integration method. The partial differential equations were transformed into a coupled set of ordinary nonlinear differential equations via a similarity transform. These nonlinear differential equations are subject to boundary conditions at the origin and at infinity. A suitable approximation of infinity was made and the solution was obtained via a quasilinearization procedure.

The following conclusions are drawn :

(1) The thickness of both the hydrodynamic and thermal boundary layers increases with the inclination to the vertical. The maximum velocity within the boundary layer decreases with the increasing inclination.

(2) For small and medium inclinations, there is good agreement between the calculated results and experiment re-

(3) Boundary layer assumptions, implying that the distance along the plate is much larger than the boundary layer thickness, are valid for the cases of small inclination.

(4) A modified empirical Nusselt number(for the vertical plate) was found having excellent correlation with numerical results presented for the cases Prandtl number less than 100. For larger Prandtl number cases, the modified Eckert equation has better correlation.

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CHAPTER I

INTRODUCTION

Free convection is the phenomena of heat and mass transfer resulting from a fluid located in a force field in the presence of a temperature gradient. The density variation with temperature causes an unbalance of the body forces resulting in fluid motion. This presentation is concerned with laminar free convection from an isothermal inclined flat surface.

The most common model previously studied is the vertical plate immersed in an initially stationary fluid. Pohlhausen (1) developed an exact solution for a laminar viscous fluid. Schmidt and Beckmann (2) experimentally established the boundary layer characteristics which agree with the analytical preditions of Pohlhausen (see reference (1)). Ostrach (3) analytically investigated the vertical flat plate for large Grashof numbers. The analysis compares favorably with Schmidt and Beckmann's temperature distribution but varies significantly from the measured velocity distributions. Albers numerically solved the vertical plate case (see Appendix B of reference (3)) by transforming the ordinary differential equations and estimating their eigenvalues. He then numerically integrated the equations. Improved estimates of the eigenvalues were then made on the basis of

the preceding runs and the process was repeated successively until a satisfactory solution was obtained. The numerical results he obtained were for the cases of Prandtl numbers ranging from 0.01 to 1000. Sparrow and Gregg have also made significant contributions in this field. They investigated the cases of uniform surface heat flux (4) and the problem of variable fluid-property free convection (5) about a vertical flat plate. The effect of other types of acceleration fields on the free convection flow about a vertical flat plate has also been studied, such as Emery's investigation (6) on the effect of a magnetic field and Lemlich's study on the problem of spatially varying acceleration about an isothermal flat plate (7).

The case of the inclined plate in a gravitational field has received little attention. An experimental study of heat transfer from an isothermal inclined flat plate was made by Rich (8). He measured the heat transfer coefficients and the temperature distributions for the case of air at angles of inclination ranging from 0 to 40 degrees measured from the vertical position and with the temperature differences from the plate surface and the surrounding of 200° F to 260° F. Rich concluded that the Nusselt number for the inclined plate can be predicted from the vertical plate by multiplying the latter a factor of $(\cos \Phi)^{1/4}$, where Φ is the angle of inclination from the vertical position. Prior to Rich's experiments, Tautz (9) had also measured the heat transfer from a inclined square plate. According to Tautz's experiments, the combined (upper and lower surfaces) heat transfer at angles greater than 45° is constant and equal to that from a vertical; at angles smaller than 45° the heat transfer can be obtained by linear interpolation between values for the horizontal and vertical positions.

In 1963, Michiyoshi (10) proposed third-degree polynomial velocity and temperature profiles for the case of free convection flow from an inclined flat plate. Approximate solutions were then obtained by making use of the Karman-Pohlhausen integral method (11). The problem of inclined flat plate has also been investigated analytically by …eans of perturbation to the boundary equations by Guinle (12).

In this presentation, a numerical study of free convection flow as well as heat transfer from an isothermal inclined plate will be made by using finite difference approximations and quasilinearization techniques for the solution of the governing differential equations.

CHAPTER II

DEVELOPMENT OF EQUATIONS

Since all of the natural convection processes involve density variations, any strict analyical study of natural convection problems should include density variations. However, most of the previous studies of natural heat convection model have neglected it. Such a simplification does not appear unreasonable for gases as well as some other commonly involved fluids. This intuitive feeling has corrorated in a formal manner by Ostrach in his paper (3). In many cases of engineering applications, especially in many processes of practical importance where the temperature difference between fluid and plate is small, the error incurred by the assumption is very small. Nevertheless, the effect of the density variations which caused by a nonuniform temperature field will be retained for the case presented.

The equations of motion for convection are generated from Navier-Stokes equations. For a two dimensional, steady state, laminar free convection flow, the general conservation equations are:

Momentum in x-direction:

$$\left[u\frac{\partial u}{\partial x} - u\frac{\partial u}{\partial y}\right]\rho = -\frac{\partial \rho}{\partial x} - \rho g \cos\phi + \mu \nabla^2 u$$

Momentum in y-direction:

$$\left[\mathcal{U}\frac{\partial \mathcal{V}}{\partial X} + \mathcal{V}\frac{\partial \mathcal{V}}{\partial y} \right] \rho = -\frac{\partial \rho}{\partial y} - \rho g \sin \phi + \mu \nabla^2 \mathcal{V}$$

Energy:

$$\left[u \frac{\partial t}{\partial X} + v \frac{\partial t}{\partial y} \right] = \alpha \nabla^2 t$$

(2-1)

Continuity:







with boundary conditions:

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where:

 $\nabla^2 = \frac{\partial^2}{\partial \chi^2} - \frac{\partial^2}{\partial \gamma^2}$ $\mu = \text{absolute viscosity, lb/ft-sec}$ $\alpha = \text{thermal diffusivity, k/Cp } \rho , \text{ft}^2/\text{sec}$ $\phi = \text{angle of inclination measured from the vertical}$ $k = \text{thermal conductivity, Btu/hr-ft}^2 - F^0$ $Cp = \text{specific heat, Btu/lb-F}^0$

The physical model and the coordinate system are shown in Figure 2-1. The subscript " ∞ " indicates the properties at the outer edge of the boundary layer, of more precisely at infinity, and subscript "w" indicates those at the surface of the plate.

According to the concept of boundary layer theory, for the case of free convection the flow is confined to a very thin layer in the immediate neighborhood of the flat plate. The thickness of this boundary layer increases along the plate in the downstream direction; Figure 2-1 represents digrammatically the velocity distribution and temperature distribution in such a boundary layer. The dimensions across it are considerably exaggerated. The fluid is considered to be still in the region characterized by $x \leq 0$.

Since the boundary layer thickness is very small

compare with any charateristic dimension of the plate, it can be expected that the change of the parameters like v, u and p characterizing the flow is much more rapid in the y direction than in the x direction. Thus an order of magnitude analysis (19) can be made for the simplification of the partial differential equations.

After making the analysis the system of equations is reduced to the form:

$$\begin{bmatrix} u \frac{\partial u}{\partial X} + u \frac{\partial u}{\partial y} \end{bmatrix} \rho = -\frac{\partial P}{\partial X} - \rho g \cos \phi + \mu \frac{\partial^2 u}{\partial y^2}$$

$$\frac{\partial P}{\partial y} = -\rho g \sin \phi$$

$$\begin{bmatrix} u \frac{\partial t}{\partial X} + v \frac{\partial t}{\partial y} \end{bmatrix} = \alpha \frac{\partial^2 t}{\partial y^2}$$

$$\frac{\partial u}{\partial X} + \frac{\partial v}{\partial y} = 0$$
(2-2)

with boundary conditions:

 $y = 0 : \mathcal{U} = 0 ; \mathcal{U} = 0 ; t = t_{\omega}$ $y \rightarrow \infty : \mathcal{U} \rightarrow 0 ; \mathcal{U} \rightarrow 0 ; t \rightarrow t_{\infty}; P \rightarrow P_{\infty}$

First equation of equations (2-2) will be rearranged by adding to and substracting from the right-hand side the term $\rho_{\infty} g \cos \phi$. Thus, it becomes:

$$\begin{bmatrix} u \frac{\partial u}{\partial X} + v \frac{\partial u}{\partial y} \end{bmatrix} \rho = -\frac{\partial P}{\partial X} + \mu \frac{\partial^2 u}{\partial X^2} - \rho_{\omega} g \cos \phi - (\rho - \rho_{\omega}) g \cos \phi \qquad (2-3)$$

In specifying the above equation at the outer edge of the boundary layer where u=0, v=0 and $\rho = \rho_{\infty}$ the following result is obtained:

$$\frac{\partial P}{\partial X} + P_{\infty} g \cos \phi = 0 \tag{2-4}$$

When the flow velocity is small compared with the sonic velocity say, a Mach number less than 0.3, the fluid involved is always considered incompressible. Thus, for the case of free convection, density variations will be assumed to be temperature dependent only. The coefficient of thermal expansion β is defined as:

$$\beta = \lim_{\Delta t \to 0} \frac{\Delta V V_i}{\Delta t}$$
(2-5)

where;

$$V_{i}$$
 = initial volume

 $\triangle V$ and $\triangle t$ are increments of volume and temperature respectively. For engineering purposes, an average can be defined as:

$$\beta = \frac{1}{V_i} \frac{\Delta V}{\Delta t} = \rho_i \frac{\frac{1}{\rho_f} - \frac{1}{\rho_i}}{t_f - t_i} = \frac{1}{\rho_f} \frac{\rho_i - \rho_f}{t_f - t_i}$$
(2-6)

where the subscript "f" stands for the final state and "i" stands for the initial state.

Equation (2-6) can be expressed as :

$$\rho_i - \rho_f = \beta \rho_f (t_f - t_i)$$
(2-7)

For the case here the above expression should be:

$$\rho - \rho_{\infty} = \beta \rho_{\omega}(t_{\infty} - t)$$
(2-8)

Thus the first equation of equation (2-2) can be rewritten as:

$$\left[u\frac{\partial \mathcal{U}}{\partial \mathcal{X}} + \mathcal{V}\frac{\partial \mathcal{U}}{\partial \mathcal{Y}}\right] = \nu \frac{\partial^2 \mathcal{U}}{\partial \mathcal{Y}^2} + \beta g \cos\phi (t - t_{\infty})$$
(2-9)

where ;

 ν = kinematic viscosity, $\mu_{/p}$, ft²/sec

It is worth-while noting that for the case of a gas, Sparrow and Gregg (5) developed a reference temperature which was defined as :

$$t_r = t_\omega - a38(t_\omega - t_\infty)$$
 (2-10)

To extend the constant property results to the variable-property situations, the suggestion was made to replace the thermal expansion coefficient β by $1/t_{\infty}$ and to evalute the other properties of the involved fluid at the defined reference temperature t_r .

From the second of equations (2-10), it is known that a pressure variation across the boundary layer (which is different from $\frac{\partial P}{\partial y} = 0$ for the case of vertical plate) does exist for the inclined plate. A more rigorous expression (see the discussion of reference (8)) should be:

$$\frac{\partial P}{\partial y} = \rho_{\omega} g \sin \phi - \frac{t - t_{\omega}}{t_{\omega}}$$
(2-11)

Thus, the governing equations become:

$$\begin{aligned}
\mathcal{U}\frac{\partial \mathcal{U}}{\partial X} + \mathcal{U}\frac{\partial \mathcal{U}}{\partial y} &= \mathcal{V}\frac{\partial^{2}\mathcal{U}}{\partial X^{2}} - \beta g \cos \phi (t - t_{\infty}) \\
\frac{\partial \mathcal{P}}{\partial y} &= \rho_{\infty}g \sin \phi \beta (t - t_{\infty}) \\
\mathcal{U}\frac{\partial t}{\partial X} + \mathcal{U}\frac{\partial t}{\partial y} &= \alpha \frac{\partial^{2}t}{\partial y^{2}} \\
\frac{\partial \mathcal{U}}{\partial X} + \frac{\partial \mathcal{U}}{\partial y} &= 0
\end{aligned}$$
(2-12)

with boundary conditions :

Thuc

$$\begin{array}{l} y = 0 : \mathcal{U} = 0 ; \mathcal{V} = 0 ; t = t_{\omega} \\ y \rightarrow \infty : \mathcal{U} \rightarrow 0 ; \mathcal{V} \rightarrow 0 ; t \rightarrow t_{\infty} ; P \rightarrow P_{\alpha} \end{array}$$

Quite a number of simplifications have been made to the orginal system of partial differential equations. However, there is still a fourth order system of equations to be solved. It is fortunate that the similarity transfromation which Pohlhausen (1) has applied to the governing partial differential equations of the case of the vertical plate can be suitably applied to this case. Furthermore, by examining the equations (2-10), it is found that the dependent variable, p, which appeared in the second equation of this system does not appear in any other equations of the system. This implies that the second equation can be treated separately in the numerical procedures.

In order to transform this system of partial

differential equations, the solution of equations (2-10) will be written in terms of a stream function Ψ , defined by the relations :

$$\mathcal{U} = \frac{\partial \Psi}{\partial y}$$
; $\mathcal{V} = -\frac{\partial \Psi}{\partial x}$ (2-13)

An independent variable η , the so-called similarity variable is also defined by:

$$\eta = C \frac{y}{\sqrt[4]{\chi}}$$
(2-14)

where ;

$$C = \left[\frac{g(t_{\omega} - t_{\infty})}{4\nu^2 t_{\infty}}\right]^{\frac{1}{4}}$$

New dependent variables f and θ are given by :

$$f(\eta) = \left[\frac{\Psi}{\chi^{3} 4}\right] \left[\frac{1}{4\nu c}\right]$$

$$\theta(\eta) = \frac{(t - t_{\infty})}{(t_{\omega} - t_{\infty})}$$
(2-16)

The velocity components now become :

$$u = 4\nu x^{\frac{1}{2}}c^{2}f(\eta)$$

$$v = \nu c x^{-\frac{1}{4}}[\eta f(\eta) - 3f(\eta)]$$
(2-17)

The partial derivatives of the parameters which characterize the flow are related to the new variables as follows :

$$\frac{\partial \eta}{\partial \chi} = -\frac{1}{4} \chi^{-1} \eta$$

$$\frac{\partial \eta}{\partial y} = c x^{-\frac{1}{4}}$$

$$\frac{\partial U}{\partial x} = v c^{2} x^{-\frac{1}{2}} (2f' - f''\eta)$$

$$\frac{\partial U}{\partial y} = 4v c^{3} x^{-\frac{1}{4}} f''$$

$$\frac{\partial^{2} U}{\partial y^{2}} = 4v^{2} c^{4} f''$$

$$\frac{\partial^{2} U}{\partial y^{2}} = v c^{2} x^{\frac{1}{2}} (f''\eta - 2f')$$

$$\frac{\partial t}{\partial y} = v c^{2} x^{-\frac{1}{2}} (f''\eta - 2f')$$

$$\frac{\partial t}{\partial y} = \theta' c x^{-\frac{1}{4}} (t_{\omega} - t_{\omega})$$

$$\frac{\partial^{2} t}{\partial y^{2}} = \theta'' c^{2} x^{-\frac{1}{2}} (t_{\omega} - t_{\omega})$$

where each prime represents the differentiation with respect to the new independent variable η .

Substituting all these partial derivatives into equations (2-12), two simultaneous nonlinear ordinary differential equations are obtained :

$$f''' + 3f''f - 2(f')^{2} + \theta \cos \phi = 0$$
(2-19)
$$\theta'' + 3P_{r}f \theta' = 0$$

where ;

Pr = Prandtl number,
$$C_p \not \mu / k$$
, dimensionless

These equations are subject to the boundary conditions :

 $\eta = 0 : f = 0 ; f' = 0 ; \theta = 1$ $\eta \rightarrow \infty : f' \rightarrow 0 ; \theta \rightarrow 0$

The pressure gradient normal to the surface of the plate is now written as :

$$\frac{\partial P}{\partial y} = P_{\omega} g \beta \theta \sin \phi \left(t_{\omega} - t_{\infty} \right) \tag{2-20}$$

with the boundary condition

$$y \rightarrow \infty : P \rightarrow P_{\infty}$$

It is observed that the pressure distribution normal to the surface can be obtained numerically by integrating the equation (2-20) together with equation (2-4) after equations (2-19) have been solved.

CHAPTER III

NUMERICAL PROCEDURES

The solution of a system of differential equations, subject to appropriate initial conditions, can generally be obtained by a forward integration method. The most common methods used for this integration are predictor-corrector methods and Runge-Kutta methods, because of their excellent stability and flexibility (13). For a system of equations including higher-order terms, a simple change of variable will usually change the system of equations into a system of first-order equations to facilitate programming.

For the boundary-value problems which do not provide a full set of initial conditions, the straight-forward application of numerical integration techniques used for initial value problems is often insufficient. If the problems are linear, superposition will yield a finite series of initial value problems whose weighted sum will satisfy the boundary conditions.

The governing simultaneous differential equations have already been derived for the case described in the preceding chapter are:

$$f''' + 3ff'' - 2f'^{2} + \theta \cos \phi = 0$$
 (2-19)
$$\theta'' + 3P_{r}f\theta' = 0$$

with boundary conditions :

$$\begin{aligned} \eta &= 0 : \quad f = 0 \quad ; \quad f' = 0 \quad ; \quad \theta = 1 \\ \eta &-\infty : \quad f' = 0 \quad ; \quad \theta = -0 \end{aligned}$$

This is a fifth-order nonlinear boundary value problem. In order to obtain an equivalent first-order system of equations, equations (2-19) will be transformed by letting :

$$f = F_{o}$$

$$f' = F_{o}' = F_{1}$$

$$f'' = F_{o}'' = F_{1}' = F_{2}$$

$$f''' = F_{o}''' = F_{1}'' = F_{2}'$$

$$= -3 f''f + 2 f'^{2} - \theta \cos \phi$$

$$\theta = F_{3}$$

$$\theta' = F_{3}'' = F_{4}$$

$$\theta'' = F_{3}'' = F_{4}'' = -3 P_{r} f \theta'$$

Therefore, the first-order equations of the system are :

$$F_{o}' = F_{1}$$

$$F_{1}' = F_{2}$$

$$F_{2}' = -3F_{2}F_{0} - 2F_{1}^{2} - F_{3}\cos\phi$$

$$F_{3}' = F_{4}$$

$$F_{4}' = -3P_{r}F_{0}F_{3}$$

(3-1)

with two-point boundary values :

$$T_1 = 0$$
 : $F_0 = 0$; $F_1 = 0$; $F_3 = 1$
 $T_1 - \infty$: $F_2 = 0$; $F_3 = 0$

As a boundary value problem, the additional difficulties are caused by the non-linearity of the system. The quasilinearization techniques(14) will circumvent this problem. All terms in the system will be approximated by their Taylors expansions. For example, the nonlinear term $-3F_2F_o$ can be approximated by :

$$-3F_{2}F_{o} \doteq -3F_{2}^{n}F_{o}^{n} - F_{o}^{n}(F_{2}^{n+1} - F_{2}^{n})$$

$$-3F_{2}^{n}(F_{o}^{n+1} - F_{o}^{n})$$
(3-2)

where subscript "n" and "n+1" indicate the nth and (n+1)st approximation. The (n+1)st approximation can be obtained after the calculation of the nth approximation.

Therefore, the linearized recurrence relationships of system are :

$$F_{o}^{'nti} = F_{1}^{n+1}$$

$$F_{1}^{'nti} = F_{2}^{n+1}$$

$$F_{2}^{'nti} = -3F_{2}^{n}F_{o}^{n} - 3F_{o}^{n}(F_{2}^{nti} - F_{2}^{n})$$

$$-3F_{2}^{n}(F_{o}^{nti} - F_{o}^{n}) + 2(F_{1}^{n})^{2}$$

$$+4F_{i}^{n}(F_{i}^{nti} - F_{1}^{n}) + F_{3}^{nti}COS\phi$$

$$F_{3}^{'nti} = F_{4}^{n+1}$$
(3-3)

$$F_{4}^{n+1} = -3Pr\left[F_{0}^{n}F_{3}^{n} + F_{0}^{n}(F_{3}^{n+1} - F_{3}^{n}) + F_{3}^{n}(F_{0}^{n+1} - F_{0}^{n})\right]$$

In matrix notation, equations (3-3) can be expressed in the following compact form :

$$\overline{F}^{n+1} = [A] \times \overline{F}^{n+1} + \overline{K}$$
(3-4)

where ;

$$\begin{bmatrix} A \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 \\ -3 F_o^n 4 F_i^n - 3 F_o^n \cos \phi & 0 \\ 0 & 0 & 0 & 0 & 1 \\ -3 P_r F_3^n & 0 & 0 & -3 P_r F_o^n & 0 \end{bmatrix}$$
(3-5)

and

$$\vec{F}^{n+1} = \begin{bmatrix} F_{0} \\ F_{1} \\ F_{2} \\ F_{3} \\ F_{4} \end{bmatrix}^{n+1} = \begin{bmatrix} 0 \\ 0 \\ 3F_{2}^{n}F_{0}^{n} - 2(F_{1}^{n})^{2} \\ 0 \\ -3PrF_{3}^{n}F_{0}^{n} \end{bmatrix}$$

The linearized equations can be solved by superposition. The general solution will be taken to have the form:

$$\overline{F} = \overline{F}_{p}^{(0)} + \sum_{j=1}^{5} \alpha_{j} \overline{F}_{h}^{(j)}$$
(3-6)

where the nonzero and zero superscripts on the right hand side represent homogeneous and particular solutions of equation (3-4) respectively. The superscript (j) indicates the jth linearly independent homogeneous solution and \propto_j are the arbitrary constants picked to ensure that F^{n+1} satisfies the boundary conditions.

A particular solution of the first approximation can be obtained by integrating equation (3-4). Because of the incompleteness of the initial conditions, a set of initial conditions has to be assumed for starting the numerical integration. This is done by assuming any finite initial conditions.

In addition to the initial conditions, and approximation of the solution has to be used in the first iteration for starting the iteration. The initial approximation of the solutions in this case will be taken as:

$$\vec{F}(\eta) = \begin{bmatrix} (1 - e^{-\eta})/2 \\ SIn(\frac{\pi}{10}\eta)e^{-\eta} \\ 0 \\ 1 - \frac{\eta}{10} \\ - \frac{1}{10} \\ 0 \end{bmatrix}$$
(3-7)

The first two elements of initial approximation (3-7) are approximated. on the basis of the f profile and the nondimensional velocity profile, f', of the solutions of the vertical plate (16). The third element is arbitrarily chosen. The last two elements are approximated by the interpolations of the two-point boundary conditions (see equations (3-3)) between $\eta = 0$ and $\eta = "\infty"$ and their derivatives.

At this point it is necessary to look at the boundary conditions at infinity. Since the boundary layer thickness is assumed to be very small. The properties of the fluid at the infinite distance from the surface of the plate will be no difference from those at a certain finite distance outside the boundary layer. Thus, " ∞ " can be replaced by a well chosen finite value. This value may not be too large. For example, $\eta = 10$ or $\eta = 15$ may satisfactorily represent infinity. The discussion and illustration of the numerical techniques for this replacing of the boundary condition at infinity is found in reference (15).

Instead of assuming finite initial condition, a set of linearly independent initial conditions have to be assumed for the linearly independent homogeneous solutions. Usually, the columns of an appropriate (of same order as the system) non-singular matrix are assumed. Simply, and identity matrix may be used as the initial conditions :

Along with the initial approximation (3-7) five homogeneous solutions $\overrightarrow{F}_{h}^{(1)}$, $\overrightarrow{F}_{h}^{(2)}$, $\overrightarrow{F}_{h}^{(3)}$, $\overrightarrow{F}_{h}^{(4)}$ and $\overrightarrow{F}_{h}^{(5)}$ can be obtained by numerically integrate the homogeneous equation of equation (3-4), that is :

$$\vec{F}(\eta) = [A] \times \vec{F}$$
(3-9)

The complete solution is then formed by adding to the particular solution the arbitrary multiplies of those five linearly independent homogeneous solutions, that is :

$$\overline{F} = \overline{F}_{P} + \sum_{j=1}^{5} \alpha_{j} \overline{F}_{h}^{(j)}$$
(3-10)

All of those solutions will be obtained as numerical data at discrete points of the range of integration. Five arbitrary constants have now to be determined to satisfy the boundary conditions. By specifying equation (3-10) at the points where the boundary conditions are given, it will result in five simultaneous equations :

At $\eta = 0$

$$F_{o}(0) = f(0) = 0 = \left[F_{o}(0)\right]_{p} + \sum_{j=1}^{5} \alpha_{j} \left[F_{o}^{(j)}(0)\right]_{h}$$

$$F_{1}(0) = f'(0) = 0 = \left[F_{1}(0)\right]_{p} + \sum_{j=1}^{5} \alpha_{j} \left[F_{1}^{(j)}(0)\right]_{h}$$

$$F_{3}(0) = \theta(0) = 1 = \left[F_{3}(0)\right]_{p} + \sum_{j=1}^{5} \alpha_{j} \left[F_{3}^{(j)}(0)\right]_{h}$$

and at $\eta = "\infty"$

$$F_{1}(\infty) = f(\infty) = 0 = \left[F_{1}(\infty)\right]_{p} + \sum_{j=1}^{5} \alpha_{j} \left[F_{1}^{(j)}(0)\right]_{h}$$

$$F_{3}(\infty) = \theta(\infty) = 0 = \left[F_{3}(\infty)\right]_{p} + \sum_{j=1}^{5} \alpha_{j} \left[F_{3}^{(j)}(0)\right]_{h}$$

or in matrix notation :

$$[B] \times \vec{\alpha} = \vec{Q} \tag{3-12}$$

where $\begin{bmatrix} B \end{bmatrix}$ is the coefficient matrix

$$\begin{bmatrix} B \end{bmatrix} = \begin{bmatrix} F_{0}^{(1)} & F_{0}^{(2)} & F_{0}^{(2)} & F_{0}^{(3)} & F_{0}^{(4)} & F_{0}^{(5)} \\ F_{1}^{(1)} & F_{1}^{(2)} & F_{1}^{(2)} & F_{1}^{(3)} & F_{1}^{(4)} & F_{1}^{(5)} \\ F_{1}^{(1)} & F_{3}^{(2)} & F_{3}^{(2)} & F_{3}^{(3)} & F_{3}^{(4)} & F_{3}^{(5)} \\ F_{1}^{(1)} & F_{1}^{(2)} & F_{1}^{(2)} & F_{1}^{(3)} & F_{1}^{(2)} & F_{1}^{(5)} \\ F_{1}^{(1)} & F_{1}^{(2)} & F_{3}^{(2)} & F_{3}^{(3)} & F_{1}^{(4)} & F_{1}^{(5)} \\ F_{3}^{(1)} & F_{3}^{(2)} & F_{3}^{(2)} & F_{3}^{(3)} & F_{1}^{(4)} & F_{1}^{(5)} \\ F_{3}^{(1)} & F_{3}^{(2)} & F_{3}^{(2)} & F_{3}^{(3)} & F_{3}^{(4)} & F_{1}^{(5)} \\ \end{bmatrix}$$

$$\vec{\alpha} = \begin{vmatrix} \alpha_{1} \\ \alpha_{2} \\ \alpha_{3} \\ \alpha_{4} \\ \alpha_{5} \end{vmatrix} \qquad ; \vec{Q} = \begin{vmatrix} -[F_{0}(0)]_{P} \\ -[F_{1}(0)]_{P} \\ 1 - [F_{4}(0)]_{P} \\ -[F_{1}(\infty)]_{P} \\ -[F_{4}(\infty)]_{P} \end{vmatrix}$$

Thus, the solution of equation (3-12) can be expressed in matrix notation as :

$$\vec{\alpha} = [B]^{-1} \times \vec{Q}$$
(3-13)

where $[B]^{-1}$ is the inverse of coefficient matrix [B].

Numerically, in order to reduce the computational round-off error Gauss-Jordan maximum pivot method (17) will be employed to solve equation (3-12) for these arbitrary constants α_i .

As an improvement, all these computational procedures will be repeated again after replacing the assumed initial . approximation with the results of preceding iteration. The iteration process will be terminated when the stable solution is reached. Accuracy of six significant figures may be obtained by terminating the iteration when the maximum absolute value of the arbitrary constants α_j is small, say, less than 10^{-6} .

For the case of air, the Prandtl number is about 0.72. Numerical solutions have been obtained by the proce-

Table 3-1 Numerical Data for Pr= 0.72, $\phi = 0^{\circ}$

77	f	<u>۲</u> ۱	<u>f</u> 11	θ	61
.000	000000	000000	•676310	1.000000	504760
.100	.003217	.062714	•578836	•949527	504641
.200	•012227	.115952	•486866	899094	503849
• 300	.026111	•160287	•400827	848799	501806
• 400	•044008	•196330	•321099	•798790	498052
.500	.065122	224729	•247988	•749257	492239
• 600	.088721	•246157	•181716	•700418	484143
•700	•114144	•261305	•122400	652508 ⁻	473660
 800 	.140796	.270870	•070048	•605766	460798
• 900	168155	•275544	•024556	•560424	445573
1.000	•195764 ·	•276003	014291	•516700	428488
1.100	•223237	•272898	046803	•474786	409519
1.200	.2502.46	•266841	073376	•434844	389090
1.300	•276526	•258406	094471	•397004	367557
1.400	•301865	•248113	110597	•361357	-•345283
1.500	•326102	•236434	122291	•327960	322626
1.600	•349119	•223785	130100	•296834	299920
1.700	•370839	·210526	134563	•267967	277468
1.300	•391215	•196966	136202	•241322	255532
1.900	•410231	•183364	135503	•216836	234332
2.000	•427893	•169929	132915	•194426	214041
2.100	•444227	•156830	128844	•1/3993	194791
2.200	• 459275	•144197	-•123547	•155430	1/56/2
2.300	•4/3086	•152128	-•11/63/	•138619	159741
2.4()0	•485721 (072/(•120588		•123441	144021
2.500	•497246	•109923	- 007169	•109774	129510
2.600	+ 5U//29	•099824	- 000149	•097499	-10(004)
2.00	• 517240 525850	081920		076661	- 092922
2 000	• 52 56 50 5 2 2 6 7 7	073932	074541	0/8801	- 082873
2.900	540638	-066499	-0.70150	-060055	073792
3.500	- 566331	-038510	043339	•032234	040536
4.000	-581001	•021663	- 025406	.017112	021798
4.500	.589375	.011948	-014402	.009028	011582
5.000	-593654	•006500	007984	.004747	006114
5-500	-596079	•003502	004359	.002491	003215
5.000	•597382	•001874	002355	•001306	001688
6.500	.598077	•000997	001262	•000685	000885
7.000	.598446	.000529	000673	•000359	000464
7.500	•598641	•000280	000357	.000188	000243
8.000	.598744	•000147	000189	.000098	000127
9.000	•598827	.000041	000053	.000027	000035
10.000	•598850	.000011	000015	.000007	000010
11.000	•598856	•000003	000004	•000002	000003
12.000	•598857	.000001	000001	.000001	000001
14.000	•598857	•000000	-•000000	•000000	000000

Table 3-2

Numerical Data for Pr= 0.72, $\varphi = 10^{\circ}$

η	f	<u>f</u> 1	f"	Θ	G'
.000	000000	000000	•668587	1.000000	502531
.100	•03181	.062016	•572584	•949720	502714
.200	.012092	•114699	•481980	899479	501933
.300	.025828	158608	•397191	•849374	499921
.400	.043541	.194344	•318588	•799552	496220
.500	.064446	.222543	•246475	•750198	490489
. 600	.087521	243864	•181067	•701529	482505
•700	.113012	258985	•122482	•653776	472160
 300 	.139434	268589	•070730	•607175	459463
• 90,0	.166568	273356	025712	•561958	444523
1.000	.193966	·273950	012778	•518339	427539
1.100	.221241	.271009	045039	•476509	408780
1.200	.248070	·265138	071456	•436631	-•388563
1.300	.274189	256898	-•092476	•398833	367238
1.400	.299388	•246806	108596	•363208	345163
1.500	•323504	235325	120340	.329913	322691
1.600	•346420	222866	128242	•298672	300152
1.700	•368057	209787	132830	.269775	277848
1.800	•388367	•196394	-•134614	•243085	256038
1.900	•407334	■182942	-•134074	•218543	234941
2.000	•424961	•169642	131649	•196067	214733
2.100	•441273	156662	127741	•175562	195545
2.200	456308	•144131	122702	156921	177470
2.300	.470117	•132148	116841	•140029	160565
2.400	•482759	120782	110421	.124768	144857
2.500	•494296	.110075	103664	.111018	130345
2.600	•504796	•100054	096751	•098660	117008
2.700	•514330	•090725	089831	•087578	104509
2.800	•522964	•082084	083020	•077662	093698
2.900	•530769	•074115	076408	•068804	
3.000	•537809	•066794	070060	•060906	074501
3.500	•563651	•038795	043434	•032780	041047
4.000	•578449	•021886	025544	•017448	022135
4.500	•586718	•012105	014524	•009229	011793
5.000	•591261	•006604	008075	•004365	006241
5.500	•593728	•003568	004421	•002560	003291
6.000	•595057	.001914	002395	•001346	001732
6.500	•595767	•001021	001287	000707	000910.
7.000	•596146	•000543	000688	•000371	000478
7.500	•596347	•000288	000366	•000195	000251
8.000	•596453	•000152	00194	•000102	000132
9•000	• 596538	.000042	000054	.000028	000036
10.000	•596562	•000012	000015	.000008	000010
11.000	•596568	.000003	000004	.000002	000003
12.000	• 596569	•000001	000001	.000001	000001
14.000	■596569	•000000	-•000000	•000000	

Table 3-3.

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	Numerical	Data for	Pr= 0.72,	$\phi = 20^{\circ}$	
,)	f	f'	fu	е	G 1
.oro	000000 -	000000	•645475	1.000000	496967
.100	.003073	•059926	•553842	•950306	496856
.200	.011688	•110940	•467296	•900650	496110
.300	•024981	.153569	•386226	.851123	494187
•400	.042141	•188380	•310978	801867	490647
• 500	.062416	•215970	•241838	•753059	485160
.600	.085115	•236960	•179012	•704907	477506
.700	.109610	251988	•122617	•657634	467577
• 800	•135336	•261699	•072669	•611467	455372
• 900	161794	•266734	•029 0 84	•566632	440988
1.100	188548	•267723	008319	•523335	424608
1.100	.215223	•265268	039811	•481768	406479
1.200	.241506	•259946	065741	•442088	-•386903
1.300	.267135	•252292	086519	•404425	366208
1.400	.291903	242799	102604	•368872	344736
1.500	•315648	.231912	114480	•335492	322827
1.600	338252	•220027	122644	•304311	300800
1.700	•359632	.207491	127591	275326	278947
1.800	.379738	•194600	129801	·248507	257525
1.900	398549	•181607	129726	•223799	236751
2.000	•416063	•168717	127786	.201129	216901
2.100	•432301	•156099	124363	180407	197810
2.200	•447297	•143883	119796	•161531	179875
2.300	●461095	.132168	114383	144394	163059
2.400	•473749	121026	108380·	128881	147393
2.500	485320	•110504	102005	·114877	132885
2.600	•495872	•100631	095437	.102266	119519
2.700	•505469	•091418	088823	•090936	107264
2.800	•514177	•082864	082280	•080778	096076
2.900	•522063	•074957	075899	•071687	085901
3.000	•529189	•067677	069750	■063565	076679
3.500	•555481	•039664	043702	. •034498	042632
4.000	•570671	•022573	025956	. 018511	023189
4.500	•579233	•012592	014896	•009869	012457
5.000	•583976	•006927	008356	•005243	006645
5.500	•586573	•003773	004614	•002780	003531
6.000	•587983	•002040	002520	•001472	001872
6.500	•588744	•001097	001366	•000779	000992
7.000	•589152	•000588	000736	•000412	000525
7.500	•589370	.000314	000395	•000218	000278
8.000	•589487	•000167	000211	•000115	000147
9.000	•589581	•000047	000060	.000032	000041
10.000	•589608	•000013	000017	•000009	000012
11.000	•589615	.000004	000005	•000003	000003
12.000	•589616	•000001	000001	•000001	000001
13.000	• 589617	.000000	000000	•000000	000000
14.000	•589616	•000000	~.000000	•000000	000000
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Table 3-4

Numerical Data for Pr= 0.72 , $\dot{\phi}=$ 30 $^{\circ}$

Ĉ	f	<u>-</u> 1	f"	0	Θ'
-000	-•000000	-•000000	•507128	1.000000	486922
•100	.002893	•056452	•522636	•951310	486820
•200	.011016	•104681	•442729	•902656	486131
• 300	.023571	•145164	•367752	854120	484354
•400	•C39808	<u>178408</u>	•298016	805835	481077
• 500	.059029	.204951	•233777	•757965	475990
•600	•080593	•225354	•175224	•710704	468881
•700	103914	.240190	•122470	•664260	459639
. 300	.128465	•250043	•075540	618848	448250
• 900	153776	.255491	•034376	•574679	434791
1.000	.179436	.257106	001171	•531954	419416
1.100	.205088	•255438	031324	•490852	402346
1.200	230431	•251011	056380	•451532	383843
1.300	•255215	•244320	076690	•414120	364222
1.400	.279235	.235819	-•092649	•378714	343780
1.500	.302332	.225921	104683	•34538C	322837
1.600	.324385	•214999	113228	•314153	301695
1.700	•345309	•203378	118726	285039	280631
1.800	.365047	191341	121605	.258016	259894
1.900	•383571	.179130	122275	.233042	239696
2.000	•400874	•166947	121120	·210053	220215
2.100	•416967	154955	118491	188970	201588
2.200	•431876	•143287	114704	.169703	183921
2.300	•445638	.132044	110038	.152151	167234
2.400	•458301	.121301	104737	.136210	151718
2.500	469917	.111111	099008	.121771	137241
2.600	•480543	.101507	093027	•108726	123847
2.700	•490239	•092509	086939	•096966	111516
2.800	♦ 499065	•084119	080861	•086388	100212
2.900	•507083	•076333	074886	•076891	089891
3.000	•514351	•C69136	069085	•068379	080500
3.500	•541401	•041151	044085	•037644	045460
4.000	•557269	023774	026632	•020482	025097
4.500	•566347	•013458	015530	•011069	013674
5.000	•571450	.007510	008846	•005959	007395
5.500	•574284	•004148	004958	.003201	003983
6.000	•575843	.002275	002747	.001718	002140
6.500	•576696	•001240	001510	.000921	001148
7.000	•577161	•000674	000825	•000494	000616
7.500	•577412	•000365	् − ∙000449	.000265	000330
8.000	•577548	.000197	`−.•000243	•000142	000177
9.000	•577661	•000057	000071	.000041	000051
10.000	•577693	•000016	000021	.000012	000015
11.000	•577702	•000005	000006	•000003	000004
13.000	•577705	•000000	000000	•000000	000000

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Numerical Data for Pr= 0.72 , ϕ = 40 $^{\circ}$

24	f	£١	f	Θ	Θ'
•000	000000	000000	• 553748	1.000000	472209
•100	• 002643	•051604	•478954	•952781	472118
•200	.010078	•095923	•408084	•905594	471508
• 300	•021598	133363	•341426	858512	469929
•400	• 036536	164358	•279240	811652	467013
• 500	.054270	189368	•221747	•765164	462475
.600	.074226	.208870	•169111	•719218	456115
•700	•095877	•223356	•121437	•674005	447821
008.	.118746	•233325	•078761	629720	437563
• 900	.142408	.239275	•041050	•586556	425394
1.000	166484	•241698	•008200	· • 544700	411433
1.100	.190646	•241072	019959	• 504323	395859
1.200	.214612	.237855	043654	•465575	378897
1.300	.238145	.232481	063164	•428581	360805
1.400	•261049	.225351	078806	.393442	341856
1.500	.283169	.216837	090923	.360229	322329
1.600	.304382	.207273	099877	•328986	302499
1.700	.324598	•196955	106035	.299731	282620
1.800	.343757	186147	109759	272456	262927
1.900	.361819	.175073	111400	•247132	- 243625
2.000	.378769	•163925	111289	223712	224888
2.100	.394607	152863	109736	202131	- 206858
2.200	409349	142017	107020	182313	- 189646
2.300	423021	131489	-103396	164171	- 173335
2.400	435660	.121361	099084	.147614	157977
2.500	.447308	.111689	094280	-132543	143602
2.600	-458014	.102515	089149	.118860	130222
2.700	- 467829	.093865	083830	.106466	117826
2.800	.476805	-085752	078440	.095263	106395
2.900	- 484997	-078177	073071	-085156	095895
3.000	• 404797	.071134	067800	.076054	
3 500	670593	0/3315	-007800	042760	- 049845
4 000	• J20J0J 527/56	025585	0275/2	022744	- 029120
4 500	6/7321	02/00/	016447	0120740	
4.500 5.000	• J47 J21 557007	• 014799	- 000593	015091	- 009640
5 500	• 5 5 5 7 0 0 5 5 6 2 0 0	•000454 004757		003036	- 006746
5.000	55200	007467	- 003107	003938	- 002600
6 500	• JJ8009 559012	•002662	-003107	002195	- 001622
7 000	• JJ 9012 550571	•001401	•001744 	•001177	- 0001422
7.000	• 557571 550201	•000321	- 000520	•000845	000/77
1.500	• JJJG01 540051	•000453	- 000000	•000301	
	• JOUUJI	•000250	- 0000298	•00019Z	- 000232
9.000	• DOUTAR	•000075		•000057	000069
10.000	• JOUZ4U	•000022	- 000027	•000017	000021
12.000	• JOUZ 52	•000007	- 000008	•000005	000006
$1 \leq 0 \leq 0$	• 30UZ30 540357	•000002	000002	•000001	000002
14.000	• 200220	•000000	~••••••••	•000000	-•000000

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Numerical Data for Pr= 0.72 , ϕ = 45°

77	f	, <u>f</u> 1	f"	· Θ	
•000	000000	000000	•521470	1.000000	462848
.100	•002491	•048665	•452397	•953717	462764
•200	.009509	•090599	•386869	•907463	462200
•300	•020398	•126167	•325142	861307	460738
•4CO	•034543	155762	•267448	815357	458036
• 500	.051366	•179799	•213986	•769750	453825
•600	.070332	•198706	•164906	•724649	447912
• 700	090951	•212930	•120307	•680228	440186
• 800 003 •	•112777	•222919	•080231	•636673	430611
• 900	.135408	•229127	•044655	•594166	419222
1.000	.158491	•231998	•013498	•552886	406121
1.100	.181711	•231969	013379	•512994	391464
1.200	.204802	•229459	-•036169	•474638	375452
1.300	•227534	•224864	055108	•437941	
1.400	• 249/18	•218557		•403004	
1.500	•271199	•210880		• 369901	321674
1.500	•291858	• 202144	-091582	• 2 2 8 6 8 1	
1.00	• > 1 1 0 0 2	•,174031 102507	- 102052	• 50 9 50 6	- • 203979
1.800	• 220207 348100	●102007 172007	-102352	• 281962 254444	- 245245
2 000	• 340109	• 1 7 2 2 7	-105022	•200444	- 227424
2.100	● J04000 380457	151270	- 106006	210204	- 210022
2.200	-3950457	-140056	-102016	•210896	= 1031/1
2 300	•JJJJ007 //08657	.130997	090018	●190744 172240	- 177076
2.400	. 421258	.121176	095309	.155200	
2.500	•432906	.111853	091075	-139832	
2.600	• 443643	.102973	086473	-125745	134279
2.700	453516	•194566	081640	•12945	121875
2.800	•462573	•086649	076688	.101339	110391
2.900	.470862	• 079229	071712	•090837	099802
3.000	•478435	•072305	066786	.081350	090074
3.500	.507216	•044674	044569	.046358	052830
4.000	•524731	•026764	028063	•026082	030235
4.500	•535116	•015696	017016	•014564	017050
5.000	•541163	•009067	010060	•008096	009532
5.500	•544637	.005181	005844	•004489	005302
6.000	•546615	•002938	003353	•002485	002941
6.500	•547733	•001656	001907	•001375	001629
7.000	•548362	•000929	001077	•000760	000901
7.500	•548714	•000519	000605	•000420	000498
8.000	•548911	•000290	000339	•000232	000275
9.000	•549081	.000089	000105	•000071	000084
10.000	•549133	•000027	000033	.000022	00026
11.000	•549149	•000008	000010	•00007	000008
12.000	•549153	•000002	000003	•000002	000002
14.000	•549154	•000000	000000	•000000	000000

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Table 3-7

	Numeric	al Data for:	Pr= 0.72 ,	$\dot{\varphi} = 60^{\circ}$	
2	Ę	<u>, <u>f</u>1</u>	£11	Θ	0 1
•000	000000	000000	•402084	1.000000	424419
•100	•001928	.037744	•353146	•957560	424360
•200	.007389	•070706	•306492	•915139	423958
•300	.015918	• 099124	•262276	872789	422914
• 400	.027071	•123248	•220639	830587	420972
.500	.040433	•143342	•181706	•788632	417929
.600	•055614	.159683	145578	•747043	413627
.700	•072254	•172554	.112329	•705952	407962
.800	.090019	182247	•082001	•665498	400879
• 900	.108607	189053	•054605	•625823	392374
1.000	.127743	•193265	•030118	•587069	382484
1.100	.147183	.195172	•008483	•549370	371292
1.200	.166710	195054	010386	•512850	358911
1.300	186135	•193183	-•026604	•477622	345486
1.400	.205297	189817	040314	•443782	331178
1.500	•224057	•185199	051674	•411410	316164
1.500	.242302	179555	060860	•380567	300626
1.700	259941	•173093	-•068058	•351296	284746
1.800	•276900	. 166003	073462	•323623	268698
1.900	.293126	•158454	077263	·297556	252647
2.000	•308581	150598	079653	•273088	236743
2.100	•323240	•142565	080816	•250198	221120
2.200	.337091	•134469	080928	.228851	205893
2.300	.350135	126409	080155	•209003	191159
2.400	•362377	.118463	'078648	•190600	176995
2.500	•373833	•110699	076546	173583	163462
2.600	•384525	.103169	073975	157885	150603
2.700	•394476	•095916	071045	143439	138447
2.300	. 403718	•088969	067854	.130172	127009
2.900	•412281	.082351	064486	•118013	116292
3.000	•420199	.076075	061013	106889	106290
3.500	•451339	•049920	043861	•064405	066308
4.000	•471451	•031718	029576	•038257	040251
4.500	•484087	•019698	019124	•022521	024013
5.000	•491872	•012034	012018	•013184	014171
5.500	•496601	•007265	007404	•007692	008308
6.000	•499444	•004347	004497	•004478	004851
6.500	•501139	•002584	002703	•002604	002826
7.000	•502144	•001528	001612	•001513	001644
7.500	•502737	•000900	000956	•000879	000955
8.000	•503086	•000528	000564	•000510	000555
9.000	•503409	.000181	000195	•000172	000187
10.000	•503519	•000061	000067	•000058	000063
11.000	•503556	•000020	000023	•000019	000021
12.000	• 503568	•000007	000008	•000006	000007
14.000	•503572	•000000	000001	•000000	000001

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Table 3-8

Numerical Data for Pr= 0.72 , $\varphi=70^\circ$

7	f	<u>, 1</u> 1	f"	Θ	eı
•000	000000	•000000	• 302416	1.000000	385968
.100	.001456	•028553	268874	•961404	385927
•200	.005601	•053822	~ 236745	•922822	385651
• 300	•012115	•075953	•206115	884289	384929
• 400	•020692	•095098	•177066	845857	383580
.500	.031041	•111421	•149673	•807599	381454
• 600	.042888	125090	•123998	•769597	378429
•700	•055976	•136280	•100092	•731946	374417
•800	•070067	•145169	•077988	•694748	369359
• 900	•084940	151939	•057705	.658110	363230
1.000	100390	•156771	•039246	622138	356033
1.100	116235	159848	•022593	•586937	347800
1.200	132308	•161349	•007715	•552610	338586
1.300	148458	•161449	005437	•519250	328471
1.400	164556	•160317	016926	•486943	317548
1.500	180486	•158117	-•026830	•455764	305928
1.600	196149	155001	035235	•425776	293729
1.700	•211461	•151116	042237	•397033	281073
1.800	•226351	•146597	047940	•369573	268087
1.900	•240763	•141568	052451	•343422	254892
2.000	254652	•136143	055878	•318597	241607
2.100	•267982	130425	058332	·295100	228342
2.200	•280730	•124506	059919	•272925	215198
2.300	•292879	•118467	060743	•252053	202266
2.400	•304422	•112379	060906	·232461	189626
2.500	•315355	•106305	~ •C6O5OO	•214116	177344
2.600	•325685	•100295	059614	•196979	165477
2.700	•335418	•094395	058328	181005	154071
2.800	•344569	•088641	056716	166148	143158
2.900	•353152	•083061	054846	152356	132764
3.000	•361187	•077678	052776	•139577	122904
3.500	•393904	•054172	041053	•088989	081659
4.000	•416346	•036539	0.29754	•055866	052677
4.500	•431304	•024057	020579	•034709	033310
5.000	•441066	•015560	013783	•021418	020788
5.500	•447340	•009933	009024	•013158	012863
6.000	•451326	•006278	005812	•008061	007916
6.500	•453837	•003938	003698	•004930	004855
7.000	•455407	•002456	002332	•003011	002971
7.500	•456383	•001524	001460	•001838	001816
8.000	•456989	•000942	000910	•001121	001109
9.000	•457592	.000357	000349	•000416	000413
10.000	•457819	•000133	000132	.000154	000154
11.000	•457903.	.000049	000050	.000057	000057
12.000	•457933	.000017	000018	.000020	000021
14.000	•457946	.000001	000002	•000002	000003

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inclination (Pr=0.72)

 $\widetilde{\boldsymbol{\omega}}$



inclination (Pr=0.72)

dures described. $\eta = 15$ was used to represent infinity and a step size 0.05 was used in computation. The numerical data f, f', f", θ and θ ' are shown in Table 3-1 to Table 3-8. The profiles of dimensionless temperature distribution, θ , and dimensionless velocity distribution, f', versus variable η are ploted in Figure 3-1 and Figure 3-2 respectively for the different angles of inclination.

The velocity components can now be calculted by following equations :

$$\mathcal{U} = 4 \nu X^{\frac{1}{2}} C^{2} f(\eta)$$

$$\mathcal{U} = \nu C X^{-\frac{1}{4}} [\eta f(\eta) - 3 f(\eta)]$$
(3-14)

for certain specified coordinates (x, y).

The method used for the numerical integration is Runge-Kutta fourth order method.

The local Nusselt number is defined as :

$$Nu_{x} \equiv \frac{hx}{K} = \frac{-x}{(t_{\omega} - t_{\omega})} \frac{\partial t}{\partial y}\Big|_{y=0}$$
(3-15)

To express $\frac{\partial T}{\partial y}$ in term of known function, $\Theta'(\eta)$, equation (2-18) is used. Thus:

$$\frac{\partial t}{\partial y} = \theta(\eta) c x^{-\frac{1}{4}} (t_{\omega} - t_{\infty})$$
(3-16)

Substitution of this expression into equation (3-15) yields the local Nusselt number as :

$$N_{u_{X}} = -C X^{\frac{3}{4}} \theta'(0) \tag{3-17}$$

A dimensionless heat transfer parameter may then be defined as :

$$\frac{N_{UX}}{CX^{3/4}} = -\theta'(0)$$
 (3-18)

The computations were also made for a wide ranges of Prandtl numbers. The nondimensional heat transfer parameters ,-0'(0), were obtained for Prandtl numbers equal to 0.01, 0.72, 0.733, 1.0, 10, 100, and 200 for the angles of inclination ranging from 0° to 40° . A step size of 0.05 was also used for all these calculations.

It is found that $\eta = 15$ can represent infinity satisfactory for the low Prandtl number cases, and $\eta = 10$ is very satisfactory for the high Prandtl number cases. For very low Prandtl cases, such as Pr = 0.01, $\eta = 20$ shall be used.

The numerical results of these nondimensional heat transfer parameters are listed in Table 3-9.

According to equation (3-17), the local Nusselt numbers could also be computed as a function of local Grashof number, Gr, defined as :

 $Gr \equiv \beta g(t_{\omega} - t_{\infty}) X^{3} / v^{2}$

Thus, equation (3-17) becomes:

$$N_{u_{x}} = -\theta(0) \left(\frac{Gr}{4}\right)^{\frac{1}{4}}$$
 (3-19)

For $\phi = 20^{\circ}$ and $\phi = 40^{\circ}$, the Nusselt numbers are ploted versus local Grashof number for some fixed Prandtl numbers and are shown in Figure 3-3 and Figure 3-4 respectively

Table 3-9

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Nondimensional Heat Transfer Parameter of Free Convection from an Inclined Isothermal Flat Plate

$\Pr \phi$	0 ⁰	10 ⁰	20 ⁰	30 ⁰	40° -	
0.01	0.084149	0.083888	0.083079	0.081753	0.079814	
0.72	9:5 04760	0.502831	0.496967	0.486922	0.472209	
0.733	0.508035	0.505985	0.500087	0.489983	0.475183	
1.0	0.567307	0.565138	0.558548	0.547257	0.530719	
10	1.170247	1.165740	1.152730	1.128815	1.094665	
100	2.196541	2.187727	2.160292	2.118180	2.053873	
200	2.633341	2.616709	2.592409	2.539658	2.462414	

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Local Nusselt Number Versus Grashof Number (Free Convection from a Heated Isothermal Flat Plate Inclined 20[°] from Vertical)



Local Nusselt Number Versus Grashof Number (Free Convection from a Heated Isothermal Flat Plate Inclined 40° from vertical)

CHAPTER IV

CONCLUSIONS

The differential equations describing the free convection heat transfer about an inclined plate have been solved by a numerical integration method. The partial differential equations were transformed into a coupled set of ordinary nonlinear differential equations via a similarity transform. These nonlinear differential equations are subject to boundary conditions at the origin and at infinity, A suitable approximation of infinity was made and the solution was the obtained via a quasilinearization procedure. The numerical solutions of these equations are very sensitive for large Prandtl numbers.

The following conclusions may be drawn from a careful study of the numerical results :

(1) Influence of angle of inclination ϕ on the flow and temperature gradient within the boundary layers is seen from the profiles in Figure 3-1 and Figure 3-2. For a fixed value of x, the thickness of both hydrodynamic and thermal boundary layers increases as ϕ increases. However, the maximum velocity within the boundary layer decreases as ϕ increases. These conclusions were expected from the physics of the problem because as the angle ϕ increases, the component of the bouyant force in the y-direction increases.

(2) For the cases of small and medium angles of inclination where the fluid is air, the agreement of the calculated Nusselt number with Rich's experimental results is good. A comparison of the Nusselt number versus the local Grashof-Prandtl number is shown in Figure 4-1 for the case of $\phi = 20^{\circ}$. It was expected that the numerical results are in error for large values of ϕ because the momentum in the y-direction was neglected.

(3) Boundary layer assumptions, implying that the distance along the plate is much larger than the boundary layer thickness, are valid for the cases of small ϕ .

(4) When the Prandtl number is less than 100, the empirical Nusselt number for the vertical plate (18) can be modified according to Rich's proposal as :

$$Nu_{X} = \left[\frac{0.676P_{r}^{\frac{1}{2}}}{(0.86I + P_{r})^{\frac{1}{4}}} \left(\frac{Gr}{4}\right)^{\frac{1}{4}}\right] (COS\phi)^{\frac{1}{4}}$$
(4-1)

which has excellent correlation with numerical results shown in preceding chapter. The local Nusselt number can be approximated by this equation within 2% for Grashof number from 10^4 to 10^8 . When the Prandtl number is larger, the approximation (4-1) is poor. For the cases of large Prandtl numbers, the modified Eckert equation (8)

$$Nu_{X} = \left[\frac{0.508 \ Pr^{\frac{1}{2}}}{(0.952 + Pr)^{\frac{1}{4}}} \ Gr^{\frac{1}{4}} \right] (COS\phi)^{\frac{1}{4}}$$
(4-2)

has better correlation.



Local Nusselt Number Versus Local Grashof-Prandtl Number (inclined 20 degrees from vertical)

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